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TEZA DE DOCTORAT

(Rezumat)

**Algoritmi de conducere pentru modele de tip
pendul-invers cu componente elastice**

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In prezenta teza de doctorat, autorul trateaza algoritmi de control pentru roboti cu componente elastice. Teza este focalizata asupra a doua modele majore: modelul pendului inversat si robotul biped cu componente elastic. Sunt determinate modelele dinamice si sunt propuse solutii de conducere.

O prima abordare se refera la modelul pendului inversat. Pe baza ecuatiilor dinamice obtinute si utilizand date experimentale de laborator, sunt studiate theoretic si verificate prin simulare cateva solutii de conducere. Sunt abordate astfel cateva tehnici de proiectare a unor controlere de tip conventional, PD sau LQR, nonlineare sau de tip intelligent-fuzzy. Pentru controlerele neliniare propuse se dezvolta tehnologii de proiectare „Sliding mode control” ierarhizate pentru care modelele matematice se prezinta ca sisteme in cascada sub-actionate.

Pe baza modelului matematic al pendulului inversat (IP), se analizeaza cateva configuratii conventionale ca : modelul acrobot, pendubot, dublu-inversor si robotul biped cu componente elastice. Acest model este studiat in cazul unei arhitecturi speciale cunoscuta in literatura ca „robotul atlet”, pentru care , elementele terminale ale picioarelor au configuratii particulare elastice, C-shaped elastic leg. Este determinat modelul echivalent al acestei arhitecturi mecanice si sunt propuse solutii de conducere bazate pe controlere liniare de tip LQR, PD sau solutii neliniare de tip „sliding mode control” ierarhizate. Parametrii optimi de acordare a acestor controlere sunt determinati prin algoritmi genetici.

Studiul dinamic al locomotiei este dezvoltat prin analiza functiilor de salt ale robotilor pasitori bipedi cu picioare „C-shaped leg”.Sunt propuse solutii clasice pentru obtinerea performantelor dorite ale functiei de salt si sunt investigate cateva controlere bazate pe utilizarea lichidelor electrorheologice (ER).

O atentie deosebita este acordata simularii traiectoriilor de miscare pentru diferite solutii de conducere si descrierii platformelor experimentale si rezultatelor testelor efectuate.

Un capitol de concluzii si de identificare a unor viitoare directii de cercetare incheie prezenta teza.

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Capitolul 1 : PENDULUL INVERS:MODEL DE BAZA IN SISTEMUL ROBOT

Modelul pendul invers (IP) este o configuratie de baza in controlul robotilor (Robege 1960) Schaefer si Cannon (1966), Furuta et al. (1991) au dezvoltat teoria acestor modele [1]. Ulterior Furuta a dezvoltat modelul pendulului dublu cu actionare rotativa [2]. [3] . Solutiile propuse au fost extinse de numerosi autori atat sub raportul performantelor mecanice cat si al sistemelor de conducere. [4], [5].

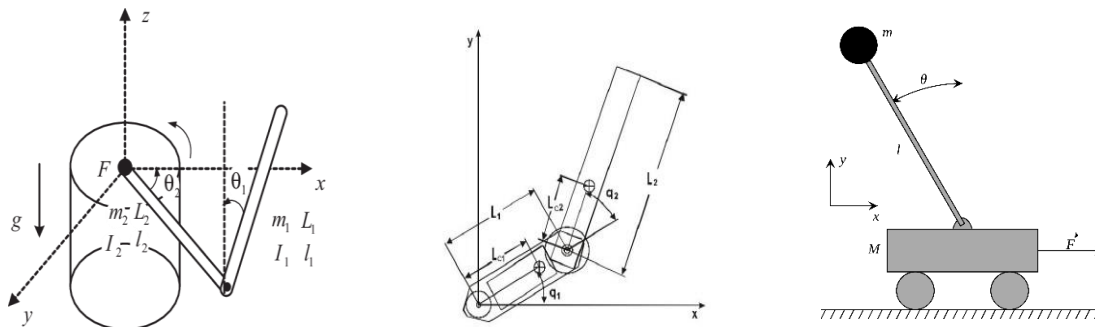


Fig 1.3 : Modele bazate pe configuratii IP

In principiu, algoritmi de control testati se bazeaza pe identificarea solutiilor de control ale pozitiiilor de echilibru in modelele IP. Au fost, de asemenea, adoptate si solutii clasice bazate pe controlere PD sau PID [64], [65], [68] sau control liniar LQR [36], [14], bazat pe tehnici consacrate de repartitie poli-zero. Avantajul acestor tehnici rezida in simplitatea lor in conditiile in care tehnicile de performanta se bazeaza pe metode de interpretare a erorii. Un interes aparte l-au jucat sistemele de conducere de tip fuzzy. Desi performantele obtinute nu sunt intotdeauna la nivelul dorit, simplitatea acestor controlere si facilitatea tehnicilor de implementare au facut ca solutiile de acest tip sa fie preferabile in multe tehnici experimentale. [69]. O clasa aparte de controlere abordeaza neliniaritatile intrinseci configuratiei mecanice prin metode „sliding mode control” [70]. O tehnica superioara rezida in structurarea unor controlere hibride ce combina atat tehnicile sistemelor inteligente cat si controlerele neliniare ierarhizate [18], [19], [59], [70].

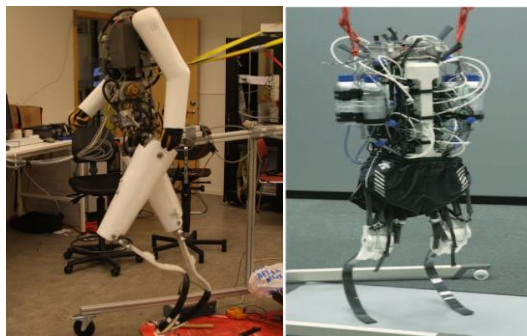


Figure 1.1: Robotul Dasher si modelul Universitatii din Tokyo

Capitolul 2: DINAMICA PENDULULUI INVERS

Modelul clasic IP a permis dezvoltarea unor configuratii cu arhitectura superioara, ce contin un numar mare de articulatii, cum ar fi pendulul dublu IP , sau structura de tip pendubot sau acrobat. Pentru toate aceste modele, obtinerea solutiilor de conducere cere determinarea cat mai exacta a modelului dinamic. In continuare vor fi analizate cateva modele de acest tip, incepand cu modelul IP on cart (Cart and Pole system) in care un carucior asigura deplasarea orizontala a sistemului rotativ al bratului.

a) Caz 1: Pendul cu masa distribuita (Fig 2.1)

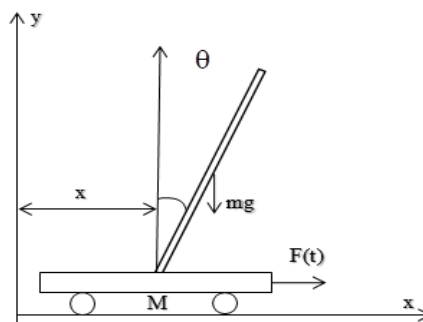


Figura 2.1: Sistemul „cart and Pole „ cu masa distribuita.

Modelul dinamic este definit ca

$$\ddot{x} = \frac{1}{\Gamma_1 + \Gamma_2 \sin^2 \theta} [\Gamma_2 \sin \theta (L\dot{\theta}^2 - g \cos \theta) + F] \quad (2.1)$$

$$\ddot{\theta} = \frac{1}{L(\Gamma_1 + \Gamma_2 \sin^2 \theta)} [-\Gamma_2 L\dot{\theta}^2 \sin \theta \cos \theta + (\Gamma_1 + \Gamma_2) g \sin \theta - F \cos \theta] \quad (2.2)$$

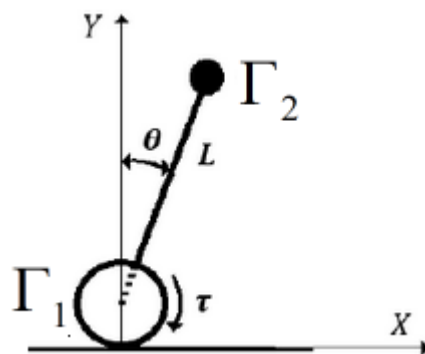


Figure 2.2: Balancing robot on wheel

Ecuatia de balans este:

$$\ddot{x} = \frac{\Gamma_2 \sin \theta (L\dot{\theta}^2 - g \cos \theta) + \tau/r}{\Gamma_1 + \Gamma_2 \sin^2 \theta} \quad (2.3)$$

$$\ddot{\theta} = \frac{-\Gamma_2 L \dot{\theta}^2 \sin \theta \cos \theta + (\Gamma_1 + \Gamma_2) g \sin \theta - \tau \cos \theta / r}{L(\Gamma_1 + \Gamma_2 \sin^2 \theta)} \quad (2.4)$$

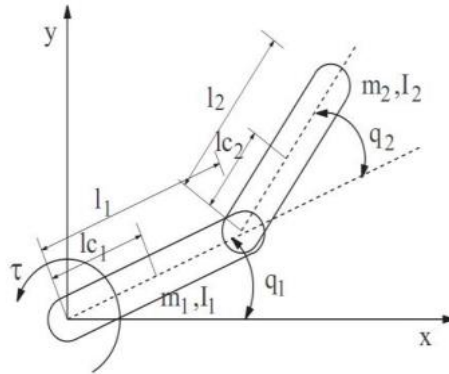


Figura 2.3: Structura unui Pendubot

Ecuatiile dinamice ale pendubot sunt

$$\ddot{q}_1 = \frac{\left[\begin{array}{l} \beta_2 \tau_1 + \beta_2 \beta_3 (x_2 + x_4)^2 \sin x_3 + \beta_3^2 x_2^2 \sin x_3 \cos x_3 + \\ -\beta_2 \beta_4 g \cos x_1 + \beta_3 \beta_5 g \cos x_3 \cos (x_1 + x_3) \end{array} \right]}{\beta_1 \beta_2 - \beta_3^2 \cos^2 x_3} \quad (2.5)$$

$$\ddot{q}_2 = \frac{\left[\begin{array}{l} (-\beta_2 - \beta_3 \cos x_3) \tau_1 + \beta_4 g (\beta_2 + \beta_3 \cos x_3) \cos x_1 + \\ -\beta_3 (\beta_2 + \beta_3 \cos x_3) (x_2 + x_4)^2 \sin x_3 + \\ -\beta_5 g (\beta_1 + \beta_3 \cos x_3) \cos (x_1 + x_3) - \beta_3 x_2^2 \sin x_3 (\beta_1 + \beta_3 \cos x_3) \end{array} \right]}{\beta_1 \beta_2 - \beta_3^2 \cos^2 x_3} \quad (2.6)$$

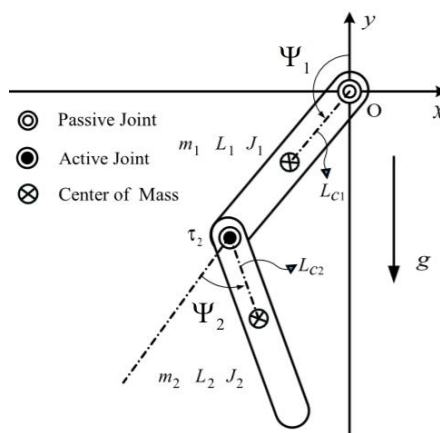


Figura 2.4: Structura unui acrobot

Neglijand frictiunea sistemului, se obtin urmatoarele ecuatii:

$$\frac{d}{dt} \left[\frac{\partial L(\Psi, \dot{\Psi})}{\partial \dot{\Psi}_i} \right] - \frac{\partial L(\Psi, \dot{\Psi})}{\partial \Psi_i} = \tau_i, (i = 1, 2) \quad (2.7)$$

Considerand $\tau_1 = 0$, (2.7) devine

$$M(\Psi_2)\ddot{\Psi} + C(\Psi, \dot{\Psi})\dot{\Psi} + G(\Psi) = \begin{bmatrix} 0 & \tau_2 \end{bmatrix}^T \quad (2.8)$$

Capitolul 3: ALGORITMI DE TIP LYAPUNOV PENTRU MODELE DE PENDUL INVERS

3.1. Metoda Lyapunov pentru modelul Cart and Pole

Teorema 3.1: Pentru modelul dynamic asociat, daca legea de control este

$$u = \frac{\sigma_1}{\varepsilon_1} x_1 + \frac{\sigma_2}{\varepsilon_2} x_2 \quad (3.1)$$

Unde coeficientii $\sigma_1 > 0$, $\sigma_2 > 0$, α , β , γ satisfac urmatoarele conditii:

$$\sigma_1 > \varepsilon_{1\max} \quad (3.2)$$

$$\sigma_1 + \sigma_2 \delta > \alpha + \varepsilon_{1\max} \beta \quad (3.3)$$

$$(\varepsilon_{1\min} - \sigma_1) \delta + \frac{1}{4} (\sigma_1 + \sigma_2 \delta - \alpha - \varepsilon_{1\max} \beta) < 0 \quad (3.4)$$

$$\delta - \sigma_1 \beta + \varepsilon_2 \beta + \varepsilon_2 \beta \eta_1 \eta_2 + \sigma_1 + \sigma_2 \delta - \alpha - \varepsilon_{1\max} \beta < 0 \quad (3.5)$$

$$\alpha > \frac{\delta}{4} > 0 \quad (3.6)$$

$$\beta > 2\delta \quad (3.7)$$

Sistemul este exponential stabil.

3.2. Control robust

Se considera modelul dynamic al sistemului IP de forma

$$\begin{bmatrix} \dot{x}_1 \\ \dot{x}_2 \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ \varepsilon_1 & -\varepsilon_2 x_1 x_2 \end{bmatrix} \begin{bmatrix} x_1 \\ x_2 \end{bmatrix} + \begin{bmatrix} 0 \\ -\varepsilon_3 \end{bmatrix} u \quad (3.8)$$

Unde restrictiile de stare de tip sector sunt definite ca

$$-\eta_1 \leq x_1 \leq \eta_1; \quad (3.9)$$

$$-\eta_2 \leq x_2 \leq \eta_2$$

Teorema 3.2: Se considera modelul IP (3.8) si legea de conducere

$$u = -ky \quad (3.10)$$

Unde rangul variabilelor este constrans de (3.9)

Daca parametrii γ , k , c_1 , c_2 , c_3 , c_4 , c_5 , c_6 satisfac conditiile

$$0 < \gamma \leq 4 \quad (3.11)$$

$$0 < k < \frac{1}{2\sqrt{\gamma}} \quad (3.12)$$

$$0 \leq \gamma + 2\operatorname{Re} \left\{ \left(\frac{C}{2} \right)^T (j\omega I - A)^{-1} B \right\} \quad (3.13)$$

Atunci sistemul este asimptotic stabil

3.3. Algoritmi fuzzy-Lyapunov pentru modele IP

Sa considera sistemul IP descries ca:

$$\dot{x} = \Delta f(x) + f(x) + (b + \Delta b)u, \quad x(0) = x_0 \quad (3.14)$$

unde: $\Delta f(x)$ si Δb reprezinta incertitudinea lui $f(x)$ si b , respectiv

Modelul fuzzy este descris de r regului fuzzy. Regula l este

If z_1 is F_{i1} and z_2 is F_{i2} and ... and z_p is F_{ip} then

$$\dot{x} = (B^i + \Delta B^i)u + (A^i + \Delta A^i)x \quad (3.15)$$

$$F_{ij}(x_i) = \begin{cases} 1 - \frac{|x_i - j_i \tilde{\Delta}|}{\tilde{\Delta}} & |x_i - j_i \tilde{\Delta}| < \tilde{\Delta} \\ 0 & \text{elsewhere} \end{cases} \quad (3.16)$$

Teorema 3.3: SE considera a lege PD si k defineste matricea de reactie. Daca urmatoarele conditii sunt verificate

a) $k_{\min} \leq k \leq k_{\max}$

b) $\operatorname{Re} \left\{ c^T (j\omega I - H)^{-1} \hat{b} \right\} + (k_{\max}^{-1} - \varepsilon^{-1} \beta) \geq 0$

unde $H = (A^1 + \nu I - k_{\min} M)$ este o matrice Hurwitz iar $\hat{b} = b^1 - d/k$

c) Perechea (H, \hat{b}) este controlabila

Atunci modelul este asimptotic stabil.

Capitolul 4: CONTROL FUZZY PENTRU MODELE DE PENDUL INVERS

4.1. Controler fuzzy bazat pe Lyapunov.

Se considera modelu IP discutat anterior si se selecteaza o functie Lyapunov de forma

$$V = \frac{1}{2}(\alpha x_1^2 + \beta x_2^2 + 2\delta x_1 x_2) \geq \frac{1}{2} \left[\left(\alpha - \frac{\delta}{2} \right) x_1^2 + (\beta - 2\delta) x_2^2 \right] \quad (4.1)$$

Derivata in raport cu timpul va fi

$$\dot{V} = \chi - \varepsilon_3 \mathcal{G} u \quad (4.2)$$

unde $\chi = 5\varepsilon_1 x_1 x_2 + 2x_2^2 - 5\varepsilon_2 x_1 x_2^3 - 2\varepsilon_2 x_1^2 x_2^2 + 2x_1 x_2$; $\mathcal{G} = 2x_1 + 5x_2$

Controlerul fuzzy va realiza conditia ca derivata (4.2) sa fie negativ definita..

Table 1: Selection condition of control signal to satisfy Lyapunov criterion

Conditia de variabila			Conditia de control
$\mathcal{G} > 0$	$x_1 x_2 > 0$	$\chi \geq 0$	$u \geq \chi / (\varepsilon_{3\min} \mathcal{G})$
		$\chi < 0$	$u \geq \chi / (\varepsilon_{3\max} \mathcal{G})$
	$x_1 x_2 < 0$	$\chi \geq 0$	$u \geq \chi / (\varepsilon_{3\min} \mathcal{G})$
		$\chi < 0$	$u \geq \chi / (\varepsilon_{3\max} \mathcal{G})$
$\mathcal{G} < 0$	$x_1 x_2 > 0$	$\chi \geq 0$	$u \leq \chi / (\varepsilon_{3\min} \mathcal{G})$
		$\chi < 0$	$u \leq \chi / (\varepsilon_{3\max} \mathcal{G})$
	$x_1 x_2 < 0$	$\chi \geq 0$	$u \leq \chi / (\varepsilon_{3\min} \mathcal{G})$
		$\chi < 0$	$u \leq \chi / (\varepsilon_{3\max} \mathcal{G})$

Funcțiile de apartenență pentru x_1 și x_2 sunt arătate în Figura 4.1 și Figure 4.2. Funcția de apartenență a ieșirii este prezentată în Fig 4.3

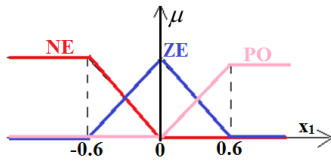


Figura 4.1: Functia de apartenenta pentru x_1

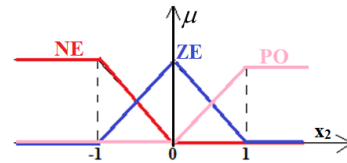


Figure 4.2: Functia de apartenenta pentru x_2

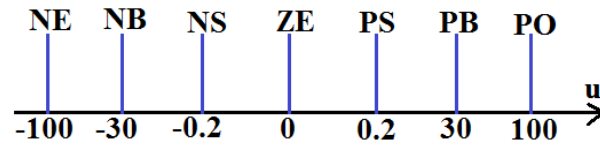


Figure 4.3: : Functia de apartenenta pentru iesire

4.2. Controler hibrid.

Sliding Mode Control reprezinta o tehnica foarte buna pentru implementarea unor controlere neliniare intr-o structura ierarhizata. [54]-[58].

Se considera modelul IP sub forma

$$\ddot{\alpha}_i = A_i(\alpha_i) + B_i(\alpha_i)u \quad (4.3)$$

unde α_i , α_{di} : sunt variabile de stare iar u : este semnalul de control

$$\text{Se defineste: } e_i = \alpha_i - \alpha_{di} \quad (4.4)$$

Eroarea de urmarire

Din (4.3), (4.4) ecuatiile echivalente vor fi

$$\ddot{e}_i = f_i + g_i u \quad (4.5)$$

Controlerul asociat lui (4.5) va stabili variabilele $e_i \xrightarrow{t \rightarrow \infty} 0$ sau $\alpha_i \xrightarrow{t \rightarrow \infty} \alpha_{id}$.

In Figura 4.4, este prezentata structura ierarhica asociata.

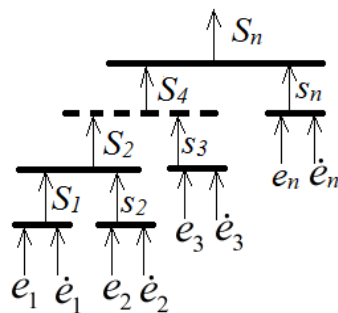


Figura 4.4: Suprafete sliding ierarhizate

Suprafetele sliding sunt

$$s_k = c_k e_k + \dot{e}_k \quad (k \leq n) \quad (4.6)$$

$$S_k = a_{k-1} S_{k-1} + s_k \quad (k \leq n) \quad (4.7)$$

unde $a_{i-1} = const$; $a_0 = S_0 = 0$.

$$S_k = \sum_{r=1}^k \left(\prod_{j=r}^k a_j \right) s_r \quad (k \leq n) \quad (4.8)$$

Pe nivelul k se obtine

$$u_k = u_{k-1} + u_{eqk} + u_{swk} \quad (k \leq n) \quad (4.9)$$

Controlul final este

$$u_n = \frac{\left[\sum_{r=1}^n \left(\prod_{j=r}^n a_j \right) b_r u_{eqr} - \eta_n \operatorname{sgn} S_n - \xi_n S_n \right]}{\sum_{r=1}^n \left(\prod_{j=r}^n a_j \right) b_r} \quad (4.10)$$

Capitolul 5: MODELE DE PENDUL INVERSE CU COMPONENTE ELASTICE

5.1. Pendulul invers elastic

In capitolele anterioare, modelul adoptat pentru pendulul elastic (E-IP) era de tipul modelului cu parametrii concentrati. In realitate, exista o distributie spatiaa a parametrilor modelului ceea ce face ca o interpretare ma exacta sa fie cea in care sistemul este descris prin ecuatii cu distributie spatiaa a variabilelor.

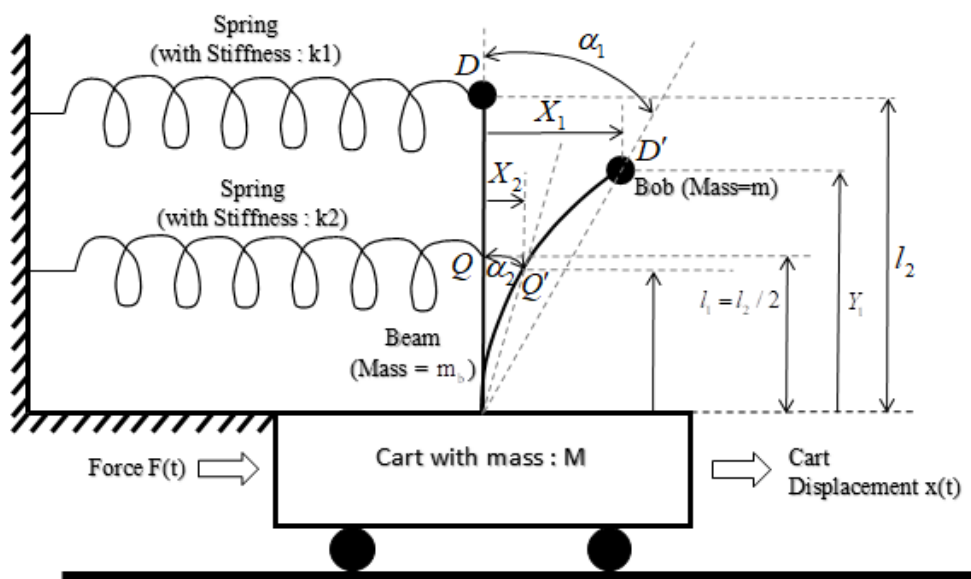


Figure 5.1: E-IP

Un astfel de model este prezentat in Fig 5.[73]-[75].

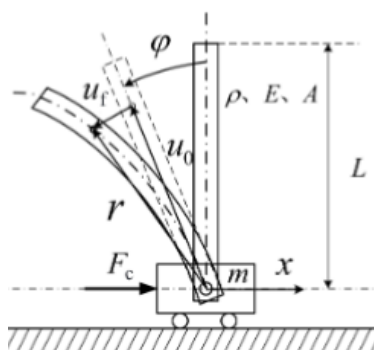


Figure 5.2: E-IP pe Cart fix

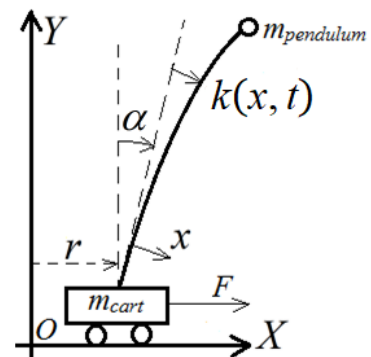


Figure 5.3: E-IP model

Conform principiului lui Hamilton

$$\int_{t_1}^{t_2} (\delta T - \delta V + \delta W_{nc}) dt = 0$$

unde δT , δV , δW_{nc} reprezinta componentele variationale ale energiilor cinetice, potentiale si lucrul fortelor neoconservative.

Modelul dinamic va fi

$$\begin{aligned} & -\left(m_{cart} + \rho l + m_{pendulum}\right) \ddot{r} - \left(m_{pendulum} l + \rho \frac{l^2}{2}\right) \ddot{\alpha} \cos \alpha + \\ & + m_{pendulum} \left[\ddot{\alpha} \times k(l,t) \sin \alpha - \dot{k}(l,t) \times \cos \alpha + \right. \\ & \left. + 2\dot{\alpha} \times \dot{k}(l,t) \sin \alpha + \dot{\alpha}^2 \times k(l,t) \times \cos \alpha \right] + \\ & + \left(\rho \frac{l^2}{2} + m_{pendulum} l \right) \dot{\alpha}^2 \sin \alpha + \rho \int_0^l \left(\ddot{\alpha} k \times \sin \alpha - \ddot{k} \times \cos \alpha + \right. \\ & \left. + 2\dot{\alpha} \dot{k} \times \sin \alpha + \dot{\alpha}^2 k \times \cos \alpha \right) dx + F = 0 \end{aligned}$$

$$\begin{aligned} & -\left(J + m_{pendulum} l^2 + \rho \frac{l^3}{3}\right) \ddot{\alpha} - \left(m_{pendulum} l + \rho \frac{l^2}{2}\right) \ddot{r} \times \cos \alpha + \\ & + m_{pendulum} \left[\ddot{r} k(l,t) \times \sin \alpha - \ddot{\alpha} k^2 \times (l,t) - l \ddot{k}(l,t) + \right. \\ & \left. - 2\dot{\alpha} \times k(l,t) \times \dot{k}(l,t) + g k(l,t) \times \cos \alpha \right] + \\ & + \rho \int_0^l \left(\ddot{r} k \times \sin \alpha - \ddot{\alpha} k^2 - \ddot{k} x - 2\dot{\alpha} \dot{k} k + g k \times \cos \alpha \right) dx + \\ & + \left(m_{pendulum} g l + \rho g \frac{l^2}{2} \right) \sin \alpha = 0 \end{aligned}$$

$$m_{pendulum} \left[k(l,t) \times \dot{\alpha}^2 - \ddot{k}(l,t) - l \ddot{\alpha} - \ddot{r} \times \cos \alpha + g \sin \alpha \right] + EI \times k'''(l,t) = 0$$

$$\rho \left(k \dot{\alpha}^2 - \ddot{k} - \ddot{\alpha} x - \ddot{r} \times \cos \alpha + g \times \sin \alpha \right) - EI \times k''' = 0$$

$$k'''(0,t) = k''(0,t) = k''(l,t) = 0$$

Se defnesc $\Psi_i(t)$, $X_i(x)$ functiile asociate modului I si $k(x,t)$ se considera ca

$k(x,t) = \sum_{i=1}^n \Psi_i(t) X_i(x)$ [76]. Considerand modul de rang 1, se obtine

$$-\left(m_{cart} + \rho l + m_{pendulum}\right) \ddot{r} - \left(m_{pendulum} l \rho \frac{l^2}{2}\right) \ddot{\alpha} \cos \alpha + m_{pendulum} \left[\begin{array}{l} X(l) \Psi \ddot{\alpha} \sin \alpha + \\ -X(l) \dot{\Psi} \cos \alpha + \\ +2X(l) \dot{\Psi} \dot{\alpha} \sin \alpha + \\ +X(l) \Psi \dot{\alpha}^2 \cos \alpha \end{array} \right] + \quad (5.6)$$

$$+\left(\rho \frac{l^2}{2} + m_{pendulum} l\right) \dot{\alpha}^2 \sin \alpha + \rho \xi_1 \Psi \ddot{\alpha} \sin \alpha - \rho \xi_1 \ddot{\Psi} \cos \alpha + 2\rho \xi_1 \dot{\Psi} \dot{\alpha} \sin \alpha + \\ +\rho \xi_1 \Psi \dot{\alpha}^2 \cos \alpha + F = 0$$

$$-\left(J + m_{pendulum} l^2 + \rho \frac{l^3}{3}\right) \ddot{\alpha} - \left(m_{pendulum} l + \rho \frac{l^2}{2}\right) \ddot{r} \cos \alpha + \rho g \xi_1 \cos \alpha \Psi + \quad 7)$$

$$+\left(m_{pendulum} g l + \rho g \frac{l^2}{2}\right) \sin \alpha + m_{pendulum} \left[\begin{array}{l} X(l) \Psi \ddot{r} \sin \alpha - X^2(l) \Psi^2 \ddot{\alpha} + \\ -X(l) \ddot{\Psi} l - 2X^2(l) \Psi \dot{\Psi} \dot{\alpha} + \\ +X(l) \Psi g \cos \alpha \end{array} \right] +$$

$$+\rho \xi_1 \ddot{r} \Psi \sin \alpha - \rho \xi_2 \Psi^2 \ddot{\alpha} - \rho \xi_3 \ddot{\Psi} - 2\rho \xi_2 \Psi \dot{\Psi} \dot{\alpha} = 0$$

$$m_{pendulum} \left[X(l) \Psi \dot{\alpha}^2 - X(l) \ddot{\Psi} - l \ddot{\alpha} - \ddot{r} \cos \alpha + g \sin \alpha \right] + \xi_4 \Psi = 0 \quad 8)$$

5.2. Modele elastice C-shaped Leg.

C-shaped leg reprezinta o structura elastica ce asigura o mai buna elasticitate a configuratiilor picioarelor



Figure 5.4: Bara curbata

$$\text{Forfecare:} \quad F_r = F \cos \theta \quad (5.9)$$

$$\text{Axial:} \quad F_\theta = F \sin \theta \quad (5.10)$$

$$\text{Moment de incovoiere:} \quad M = FR \sin \theta \quad (5.11)$$

Energia momentului M este

$$U_1 = \int \frac{M^2}{2AeE} d\theta \quad (5.12)$$

unde e este excentricitatea $e = R - r_n$. Daca $\frac{R}{h} > 10$, rezulta

$$U_1 = \int \frac{M^2 R}{2EI} d\theta \quad (5.13)$$

Energia datorata fortei axiale F_θ este

$$U_2 = \int \frac{F_\theta^2 R}{2AE} d\theta \quad (5.14)$$

Energia fortei F

$$U_4 = C \int \frac{F_r^2 R}{2AG} d\theta \quad (5.15)$$

Conform legii lui Hook

$$F = k\delta \quad (5.16)$$

unde k este constanta elastica echivalenta iar δ este deflexia..

Pentru C-shaped leg, parametrul k este variabil in lungul arcului de curba.

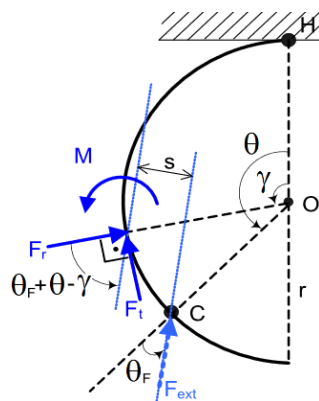


Figura 5.5: Secțiune transversala in C-shaped leg

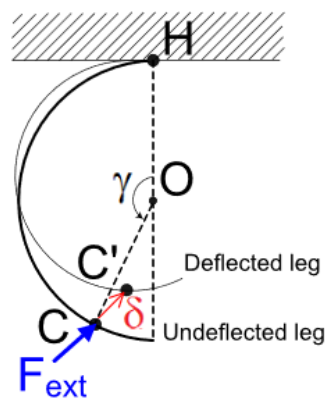
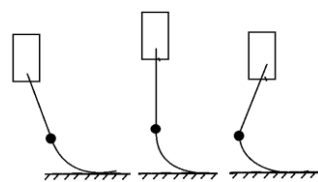


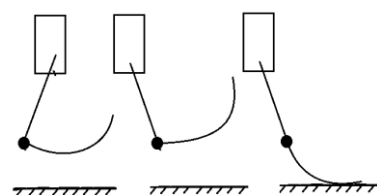
Figura 5.6: efectul fortei externe in C-shaped leg

5.3. Controlul unui Robot C-shaped Leg prin metode Lyapunov

Analizand miscarea unui robot cu doua picioare, distingem doua faze: stance and flight phase. In stance phase, picioarele sunt in contact cu solul, ifaza flight, picioarele nu au contact cu solul.



Stance phase



Flight phase

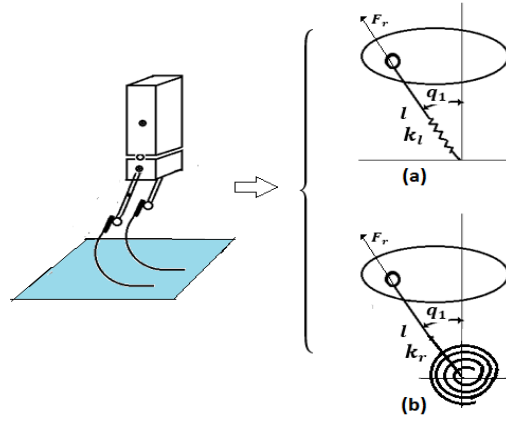


Figure 5.7: Modelul IP al unui robot cu picioare C-shaped leg:
a/ linear model/b)rotational model

Deflectia rotatională este

$$\delta_r \approx \frac{\partial U}{\partial M} \quad (5.17)$$

Se obține

$$k_l \approx \frac{M}{\delta_l} = \frac{4EI}{R^2 \left\{ \pi + \frac{1}{2} [\sin(2\varphi) + \sin(2[\pi - \varphi])] \right\}}$$

Modelul dinamic va fi

$$(M^* l^2 + I^*) \ddot{q}_1 = M^* g l \sin q_1 - k_r(\varphi) q_1 + \tau_1^* + h(\tau_2, \tau_3, q_2, q_3) \quad (5.18)$$

Se propune un controler PD

$$\tau = -\mu_1 q_1 - \mu_2 \dot{q}_1 \quad (5.19)$$

Teorema 5.1: Dacă sistemul (5.19) este supus legii de control (5.20) iar parametrii de control verifică

$$\sqrt{\frac{k_r}{N}} > \alpha > 0, \quad \eta > 0 \quad (5.20)$$

$$\mu_1 > 0; \quad \mu_2 > 0 \quad (5.21)$$

$$\begin{bmatrix} \mu_2 - \alpha N - \eta & \frac{1}{2}(\mu_2 + \alpha \mu_2 - mgl) \\ \frac{1}{2}(\mu_1 + \alpha \mu_2 - mgl) & \alpha(k_{r\max} + \mu_1 - mgl - \eta) \end{bmatrix} > 0 \quad (5.22)$$

Sistemul este asymptotic stabil.

Capitolul 6: ALGORITMI DE CONTROL AL MISCARII DE SALT

Sistemul din Fig 6.1 este format din doua picioare articulate in modul C-shaped leg in care sistemul de actionare este caracterizat de:

- Partea inferioara cu actionare hibrida electro hidraulica/pneumatica cu fluid ER.
- Componenta superioara cu actionare electrica conventionala.

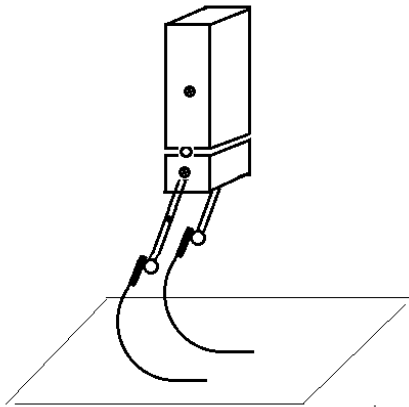


Figura 6.1: Modelul unui robot de salt

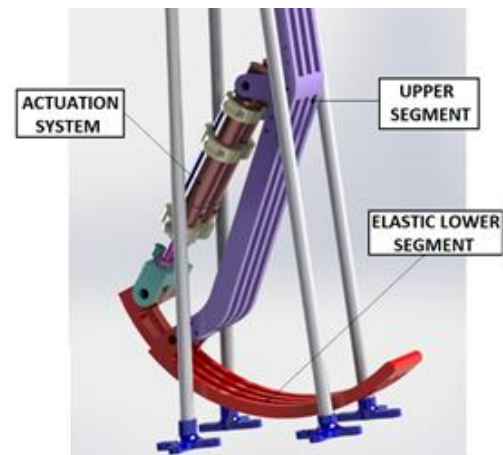


Figure 6.2: Platforma robotului de salt

Miscarea robotului este determinata de cele doua faze: faza stance cand piciorul este in contact cu solul si faza flight cand acesta paraseste solul. Frontiera intre cele doua faze este delimitata de secventele: touch-down , cand se obtine primul contact cu solul , si take-off, cand piciorul se desprinde de sol. Aceste secvente se executa periodic in cadrul ciclului de miscare.

6.1. Modelul Stance Phase

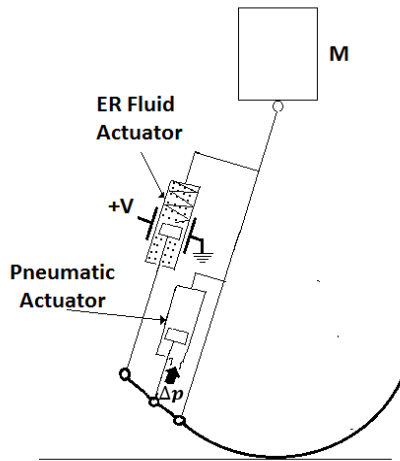


Figure 6.3: mechanical structure of leg for jumping robot

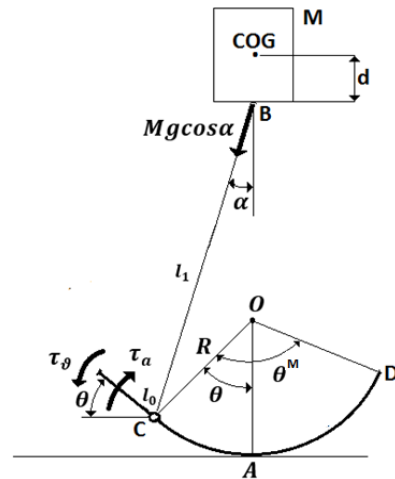


Figure 6.4: mathematical structure of leg for jumping robot

Modelul dinamic este

$$M(d+l_1)^2 \ddot{\theta} + MgR \sin \theta + 2EI\theta + \tau_g = \tau_a - \mathcal{G}_0 \dot{\theta} \quad (6.1)$$

Cu conditia initiala

$$\theta(0) = \theta_0 \quad (6.2)$$

sau

$$M(d+l_1)^2 \ddot{\theta} + J\ddot{\theta} + \mathcal{G}_0 \dot{\theta} + K_e \theta = \tau_a - \tau_g \quad (6.3)$$

Unde coeficientul dinamic echivalent este

$$K_e = MgR + 2EI \quad (6.4)$$

6.2. Secventa Stance Phase: Touch-Down

Calitatea miscarii determinata de cativa coeficienti de performanta constituie o cerinta primordiala in aceasta faza. Contactul cu solul determina oscilatii ale intregului sistem ceea ce impune gasirea unor metode adecvate de ameliorare a indicilor de calitate. In acest scop, sistemul de amortizare este prevazut cu un sistem hydraulic cu lichid ER iar investigarea regimului de miscare este bazata pe tehnici de tip skyhook. (fig 6.5-6.7)

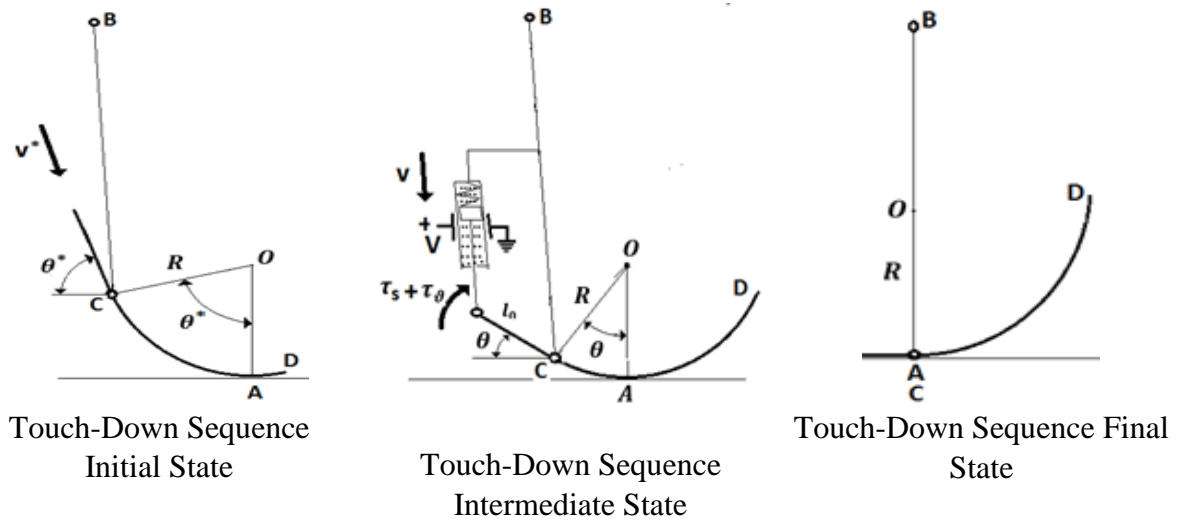


Figure 6.5: Touch-Down Sequence

Case 1: Actuator ca sistem passive damper

Modelul Touch-Down este ilustrat in **Error! Reference source not found.** unde K_f , K_s definesc coeficientii elastici ai piciorului si resortului..

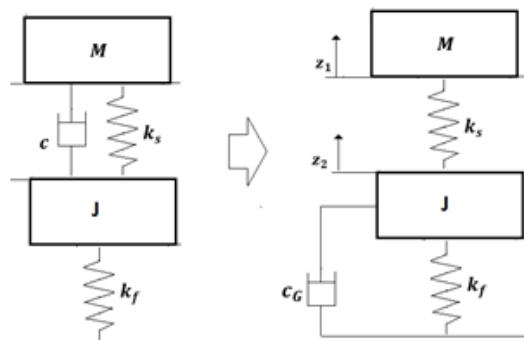


Figure 6.6: Ground-hook damper model

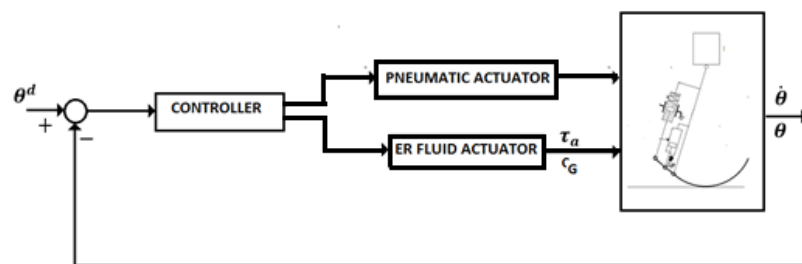


Figure 6.7: Sistemul de control al secventei Touch-Down

Modelul dinamic este

$$\ddot{\theta}J = -g_0\dot{\theta} - MgR\sin\theta - 2EI\theta + K_s(z_1 - z_2)R^* + c(\dot{z}_1 - \dot{z}_2) \quad (6.5)$$

unde z_1 , z_2 reprezinta coordonatele verticale ale celor doua conexiuni (B, C); K_s

si R^* sunt coeficientii elastici si raza echivalenta de miscare

$$R^* = l_0 \cos \theta + R \sin \theta \quad (6.6)$$

unde c este coeficient de amortizare pasiv.

Transmisibilitatea sistemului este [96].

$$T(\omega) = z_2(\omega)/z_1(\omega) \quad (6.7)$$

unde

$$z_1 = l_0 \sin \theta + R(1 - \cos \theta) + l_1 \cos \alpha \quad (6.8)$$

$$z_2 = l_0 \sin \theta + R(1 - \cos \theta) \quad (6.9)$$

Presupunand oscilatii mici in jurul punctului de echilibru

$$\left| \frac{R\theta}{l_0} \right| < 1 \quad (6.10)$$

atunci

$$z_2 \approx l_0 \theta \quad (6.11)$$

$$R^* \approx l_0 \quad (6.12)$$

Substituind **Error! Reference source not found.**, **Error! Reference source not found.** in **Error! Reference source not found.**, se obtine

$$\ddot{z}_2 = -\frac{\mathcal{G}_0 + c_G}{J} \dot{z}_2 - \frac{MgR + 2EI + K_s l_0^2}{J} z_2 + \frac{K_s l_0^2}{J} z_1 + \frac{c l_0^2}{J} \dot{z}_1 \quad (6.13)$$

Aplicand transformarea Laplace rezulta

$$T(s) = \frac{z_2(s)}{z_1(s)} = \frac{K_s l_0^2 + \frac{c l_0^2}{J} s}{s^2 + \frac{\mathcal{G}_0 + c_G}{J} s + \frac{MgR + 2EI + K_s l_0^2}{J}} \quad (6.14)$$

Substiting $s = j\omega$ in **Error! Reference source not found.**, se obtine

$$T(j\omega) = \frac{z_2(j\omega)}{z_1(j\omega)} = \frac{1 + 2\zeta(j\omega/\omega_n)}{-1 + (\omega/\omega_n)^2 + 2\zeta(\omega/\omega_n)j} \quad (6.15)$$

unde ω_n este frecventa naturala a sistemului

$$\omega_n = \sqrt{\frac{MgR + 2EI + K_s l_0^2}{J}} \quad (6.16)$$

iar ζ factorul de amortizare pasiv

$$\zeta_p = \frac{\mathcal{G}_0 + c}{2\sqrt{J(MgR + 2EI + K_s l_0^2)}} \quad (6.17)$$

Case 2: Actuator ca damper semiactiv (ground system)

O strategie ‘‘Groundhook’’ (in **Error! Reference source not found.**) se propune pentru studiul regimului oscilator. Se considera un coeficient

$$c_G = \begin{cases} c_{\max} \dot{z}_2 & \text{if } -\dot{z}_2 (\dot{z}_1 - \dot{z}_2) > 0 \\ c_{\min} \dot{z}_2 & \text{if } -\dot{z}_2 (\dot{z}_1 - \dot{z}_2) < 0 \end{cases} \quad (6.18)$$

Se obtine

$$\ddot{z}_2 = -\frac{\mathcal{G}_0 + c_G}{J} \dot{z}_2 - \frac{MgR + 2EI + K_s l_0^2}{J} z_2 + \frac{K_s l_0^2}{J} z_1 \quad (6.19)$$

sau

$$T(s) = \frac{z_2(s)}{z_1(s)} = \frac{K_s l_0^2}{s^2 + \frac{\mathcal{G}_0 + c_G}{J} s + \frac{MgR + 2EI + K_s l_0^2}{J}} \quad (6.20)$$

$$T(j\omega) = \frac{z_2(j\omega)}{z_1(j\omega)} = \frac{\eta}{-1 + \left(\frac{\omega}{\omega_n}\right)^2 + 2\zeta_G \frac{\omega}{\omega_n} j} \quad (6.21)$$

unde ω_n si ζ_G sunt definiti ca

$$\zeta_G = \frac{\mathcal{G}_0 + c_G}{2\sqrt{J(MgR + 2EI + K_s l_0^2)}} \quad (6.22)$$

$$\eta = \frac{K_s l_0^2}{MgR + 2EI + K_s l_0^2} \quad (6.23)$$

Caz 3: Actuator ca sistem ER Driver

Dinamica actuatorului este

$$\ddot{\theta} J = -\mathcal{G}_0 \dot{\theta} - MgR \sin \theta - 2EI \theta + K_s (z_1 - z_2) R^* + c_g (\dot{z}_1 - \dot{z}_2) + \tau_a \quad (6.24)$$

sau

$$\ddot{\theta} = -\frac{\mathcal{G}_0 + c_g l_0^2}{J} \dot{\theta} - \frac{MgR + 2EI + K_s l_0^2}{J} \theta + \frac{K_s l_0}{J} z_1 + \frac{c_g l_0}{J} \dot{z}_1 + \frac{1}{J} \tau_a \quad (6.25)$$

Se definesc variabilele de stare

$$\begin{cases} x = [x_1 & x_2]^T = [\theta & \dot{\theta}]^T \\ z = [z_1 & z_2]^T \end{cases} \quad (6.26)$$

Dinamica sistemului devine

$$\dot{x} = Ax + b\tau_a + Dz \quad (6.27)$$

$$y = c^T x \quad (6.28)$$

unde

$$A = \begin{bmatrix} 0 & 1 \\ -\frac{MgR + 2EI + K_s l_0^2}{J} & -\frac{g_0 + c_g l_0^2}{J} \end{bmatrix}; \quad b = \begin{bmatrix} 0 \\ \frac{1}{J} \end{bmatrix}; \quad D = \begin{bmatrix} 0 & 0 \\ \frac{K_s l_0}{J} & \frac{c_g l_0}{J} \end{bmatrix}$$

Perturbarea este evaluata in termeni de variabile de stare ca

$$z_1 = \alpha\theta \quad (\alpha < \alpha^*) \quad (6.30)$$

$$z_2 = \beta\dot{\theta} \quad (\beta < \beta^*) \quad (6.31)$$

und α , β sunt constante pozitive. mOdelul dinamic (6.27) devine

$$\dot{x} = A^* x + b\tau_a \quad (6.32)$$

unde

$$A^* = \begin{bmatrix} 0 & 1 \\ -\frac{MgR + 2EI + K_s l_0^2}{J} + \alpha & -\frac{g_0 + c_g l_0^2}{J} + \beta \end{bmatrix} \quad (6.33)$$

Se propune o lege de conducere

$$\tau_a = -ky \quad (6.34)$$

unde $k = const > 0$ satisface o conditie de sector

$$k_{\min} \leq k \leq k_{\max} \quad (6.35)$$

Teorema 6.1: Starea $[x_1 \quad x_2]^T = [\theta \quad \dot{\theta}]^T$ converge la 0 daca urmatoarele conditii sunt satisfacute:

- Matrice $H = A^* - E$ este Hurwitz, unde $E = ec^T$ este o matrice simetrica.
- (H, b) este controlabila si (H, c) este observabila.

$$c) \operatorname{Re} \left\{ \frac{c^T}{2} (sI - H)^{-1} (b - ek^{-1}) \right\} + k^{-1} \geq 0 \quad (6.36)$$

Remark 6.1:

Se definește funcția de transfer $G(s)$

$$G(s) = \frac{c^T}{2} (sI - H)^{-1} (b - ek^{-1}) \quad (6.37)$$

Considerând **Error! Reference source not found.** și condiția c) a Teoremei 6.1 se obține criteriul cercului [98]

$$\operatorname{Re} \left(\frac{k_{\max}^{-1} + G(j\omega)}{k_{\min}^{-1} + G(j\omega)} \right) > 0 \quad (6.38)$$

6.3. Secvența Stance Phase: Take-off

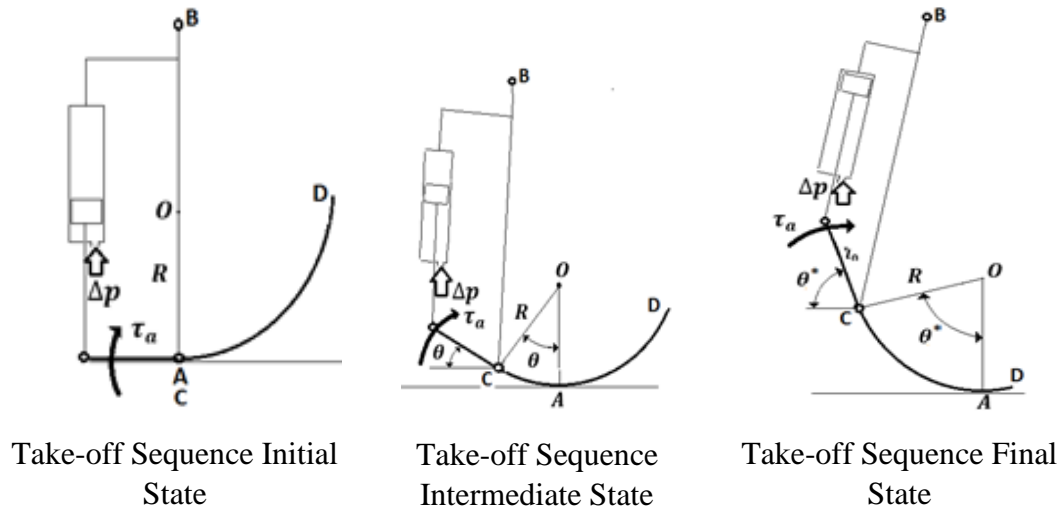


Figura 6.8: Secvența Take-off

În timpul acestei secvențe, actuatorul dezvoltă suficientă energie pentru a asigura evoluția pe traiectorie

$$W(t) \geq W^* \quad (6.39)$$

$$\frac{dW(t)}{dt} \geq \gamma > 0 \quad (6.40)$$

unde $W(0) = 0$ iar W^* este energia critică ce determină evoluția pe traiectorie, $\gamma = \text{const} > 0 \dots$

Se definește v^* viteza de start pe traiectorie la $\theta = \theta^*$ (în **Error! Reference source not found.c**). Energia critică va fi

$$W^* = \frac{1}{2} K_f (\theta^*)^2 + \frac{1}{2I_0^2} J (\dot{\theta}^*)^2 \quad (6.41)$$

Iar energia totala

$$W = w^T w = \frac{1}{2} K_f \theta^2 + \frac{1}{2} J \dot{\theta}^2 \quad (6.42)$$

Unde primul termen corespunde energiei elastice inmagazinata in picior.

$$w = \begin{bmatrix} \frac{\theta}{\sqrt{\frac{2}{K_e}}} & \frac{\dot{\theta}}{\sqrt{\frac{2}{J}}} \end{bmatrix}^T$$

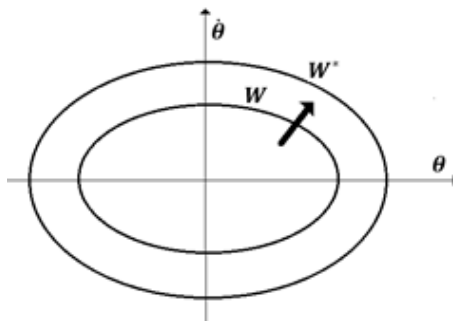


Figura 6.9: Elipsoid de energie

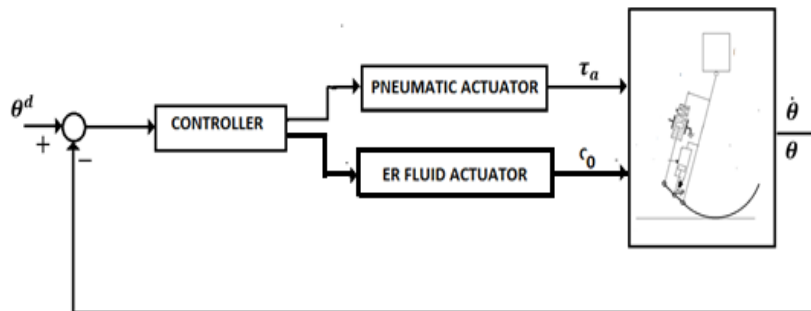


Figure 6.10: Controlul secventei Take-off

Theorem 6.2: Condițiile de salt **Error! Reference source not found.**) și **Error! Reference source not found.**) sunt satisfăcute dacă legea de control este

$$\tau_a = -k_1^J \theta + k_2^J \dot{\theta} \quad (6.44)$$

unde k_1^J , k_2^J sunt constante pozitive ce satisfac

$$k_1^J > K_f - K_e \quad (6.45)$$

$$2k_2^J - k_1^J > 2(\vartheta_0 + c_0) - K_f + K_e \quad (6.46)$$

Capitolul 7: SIMULAREA ALGORITMILOR DE CONTROL

7.1. Simularea controlului LQR pe modelul E-IP.

Se considera modelul E-IP si un controler LQR in care parametrii matricilor sunt selectati prin tehnici GA. Rezultatele simularii sunt prezentate in Fig7.1, Fig 7.2

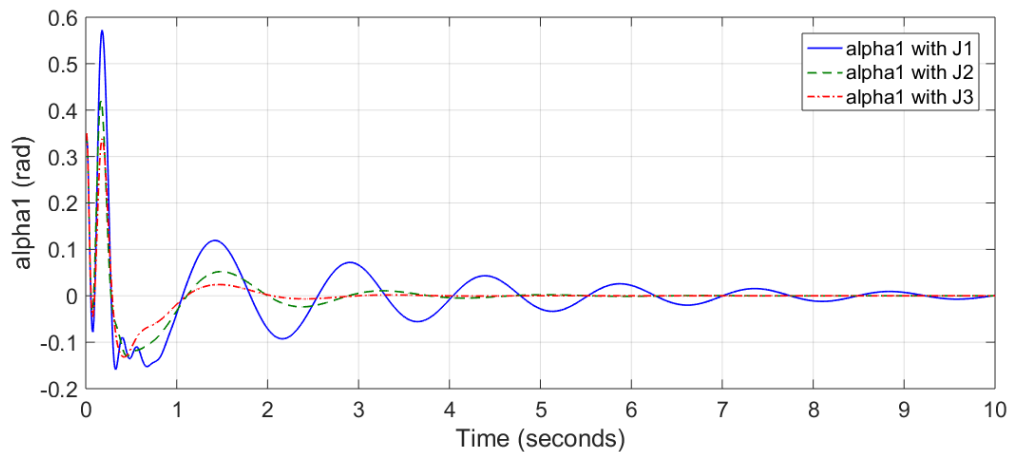


Figura7.1: Comparatia raspunsurilor modelului E-IP prin controler LQR pentru α_1 (rad)

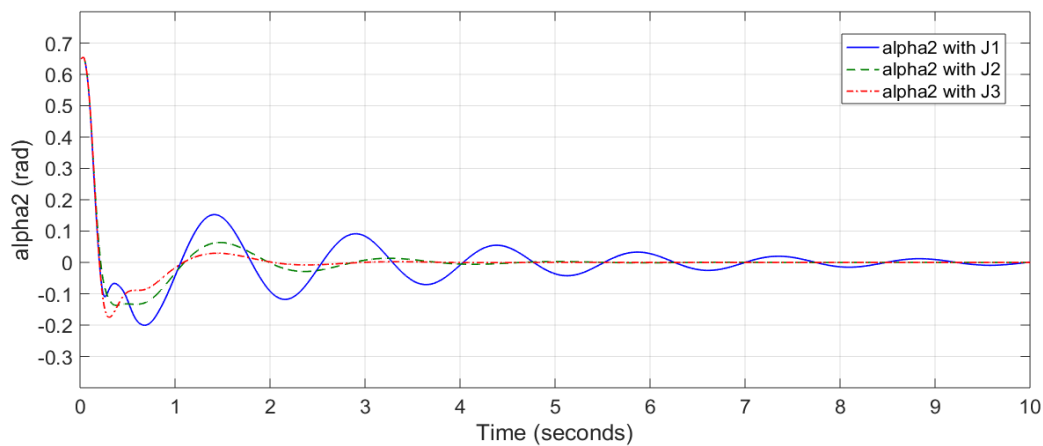


Figure 7.2: Comparatia raspunsurilor modelului E-IP prin controler LQR pentru α_2 (rad)

7.2. Control HSM pentru sistem E-IP

Se considera un control

$$u = \frac{(a_1 a_2 g_1 u_{eq1} + a_2 g_2 u_{eq2} + g_3 u_{eq3}) - (k_3 S_3 + \eta_3 \text{sign} S_3)}{a_1 a_2 g_1 + a_2 g_2 + g_3}$$

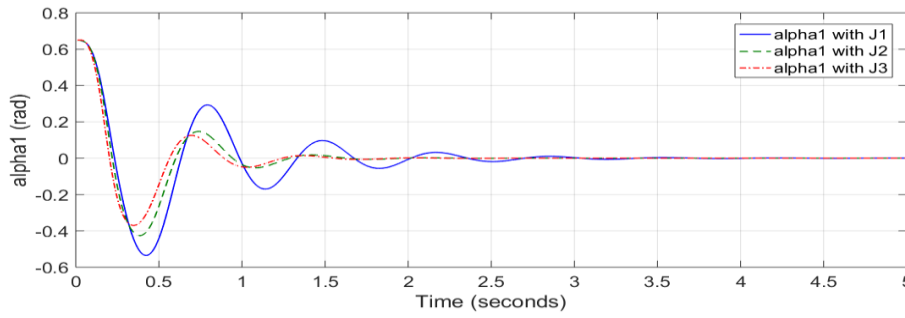


Figure 7.3: Comparatia raspunsurilor modelului E-IP prin controler HSM pentru α_1 (rad)

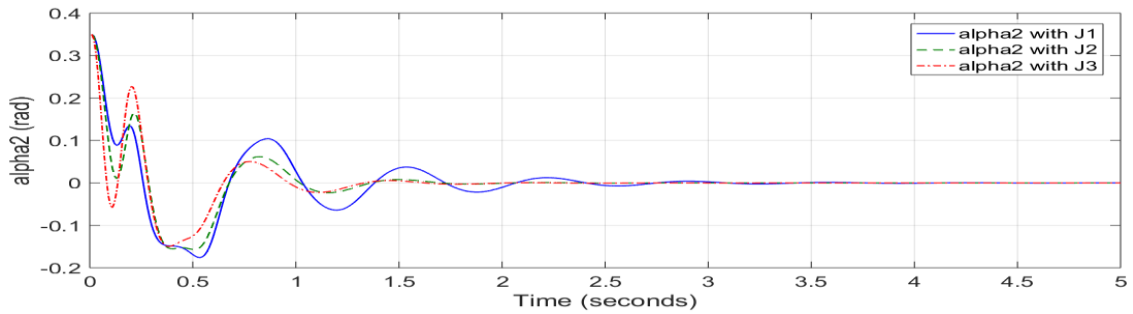


Figure 7.4: Comparatia raspunsurilor modelului E-IP prin controler HSM pentru α_2 (rad)

7.3. Control Conventional PD pentru robot biped.

Se considera un control Pd pentru robotul biped analizat (Fig 7.5).

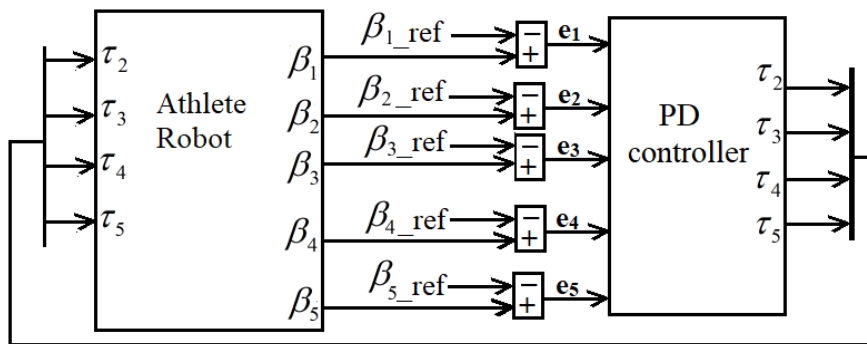


Figura 7.5: Sistemul de control

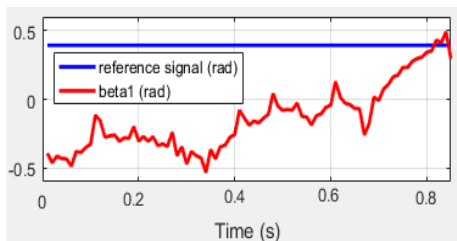


Figure 7.6: Reference signal β_{1_ref} and β_1

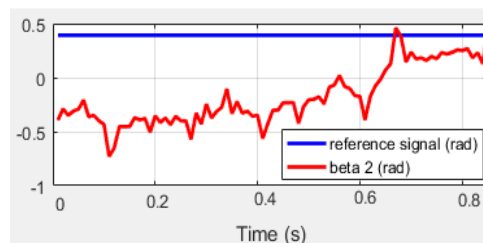


Figure 7.7: Reference signal β_{2_ref} and β_2

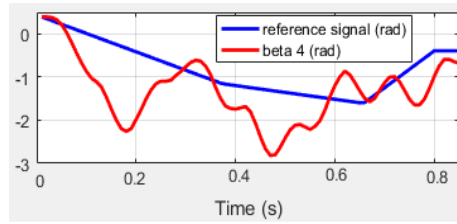
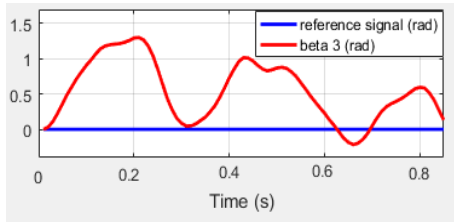


Figure 7.8: Reference signal β_{3_ref} and β_3 Figure 7.9: Reference signal β_{4_ref} and β_4

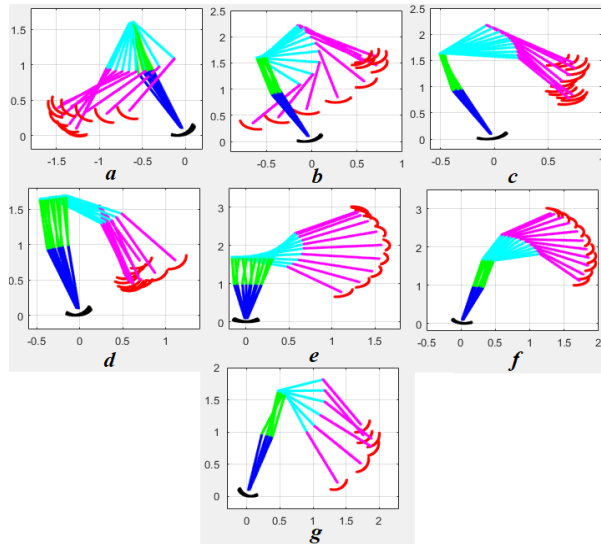
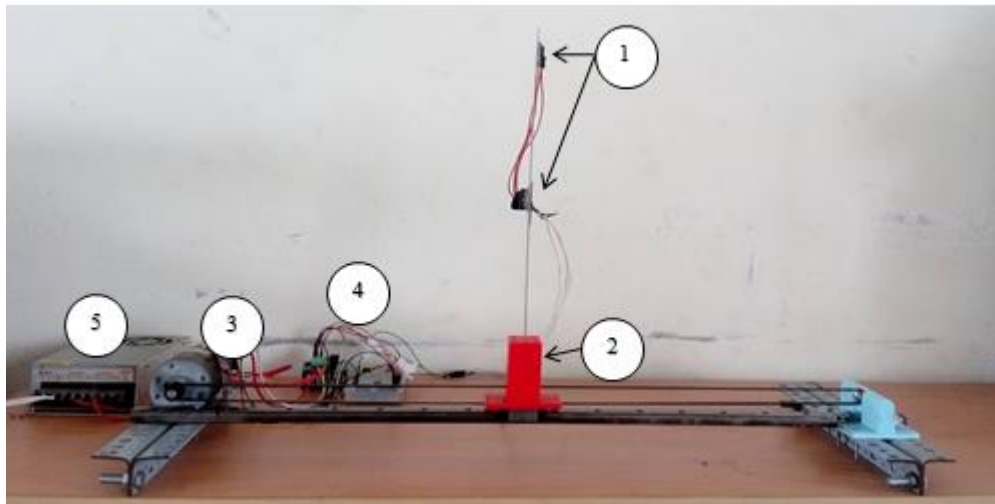


Figure 7.10: Miscarea robotului AR

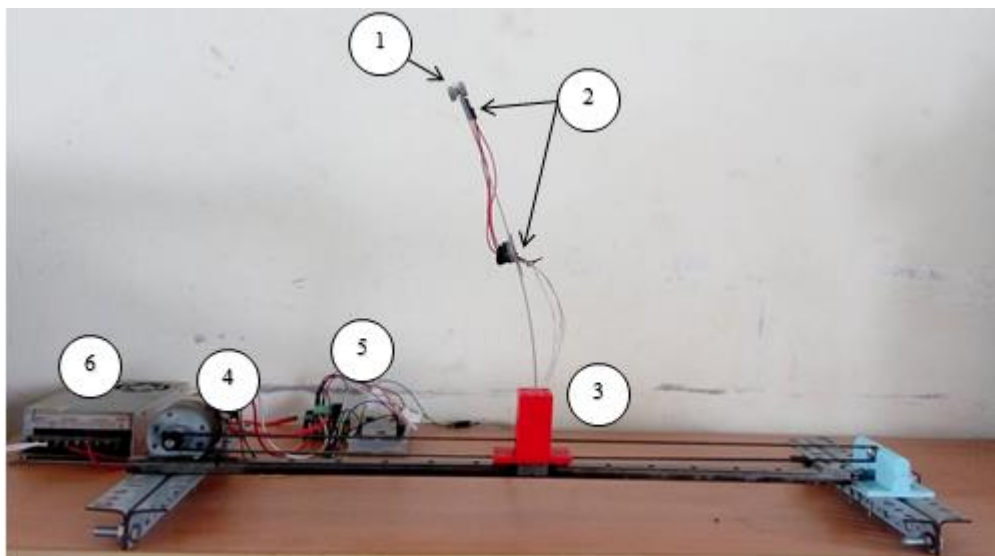
Capitolul 8: STUDIU EXPERIMENTAL AL MODELELOR CU COMPONENTE ELASTICE

8.1. Pendulul elastic invers

Platforma experimentală este prezentată în Fig 8.1



(a)



(b)

Figure 8.1: Platforma experimentală E-IP

8.2. Robot biped cu picioare elastice

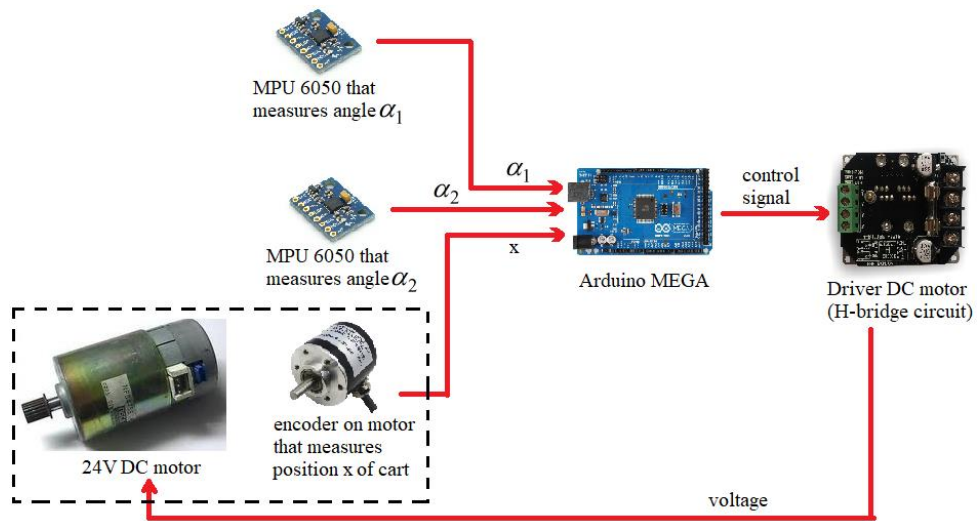


Figure 8.2 Sistemul electronic

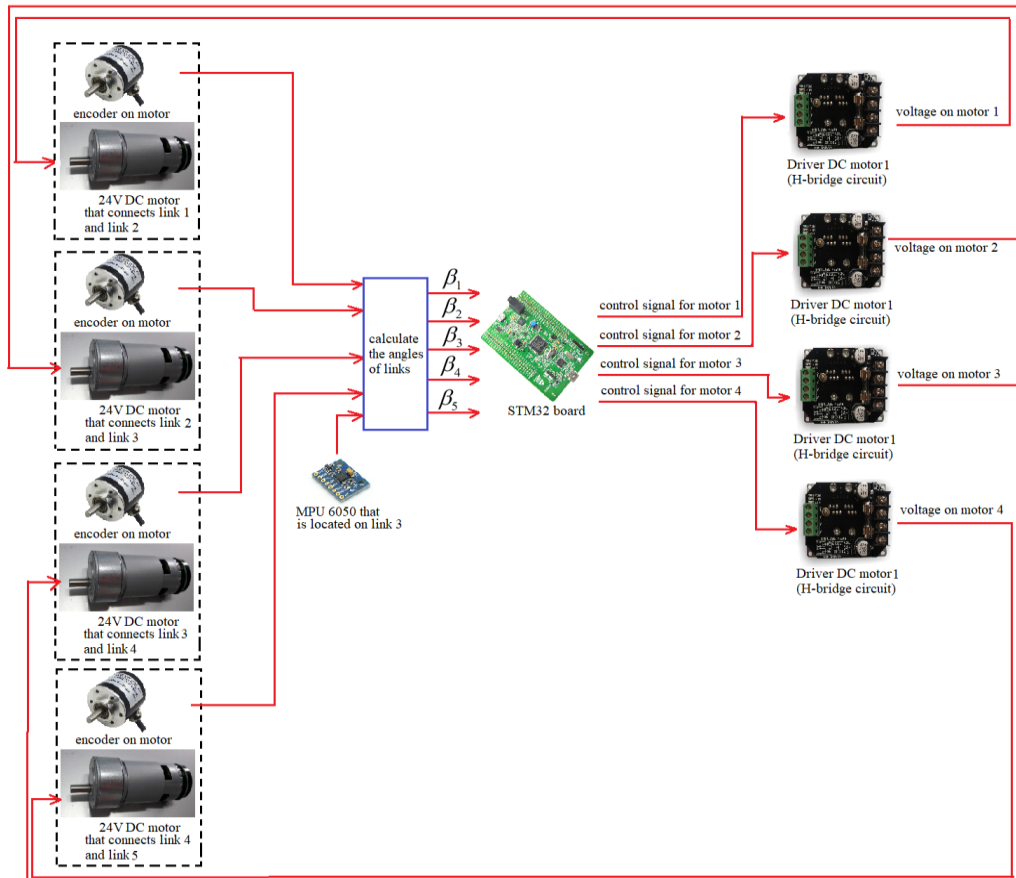


Figure 8.3: Structura hardware

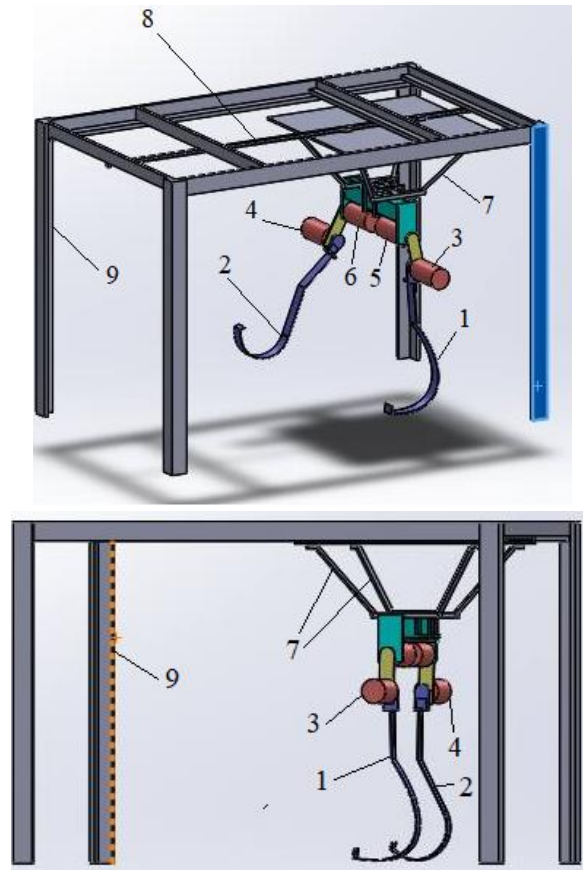


Figure 8.4: Model experimental in Solidworks

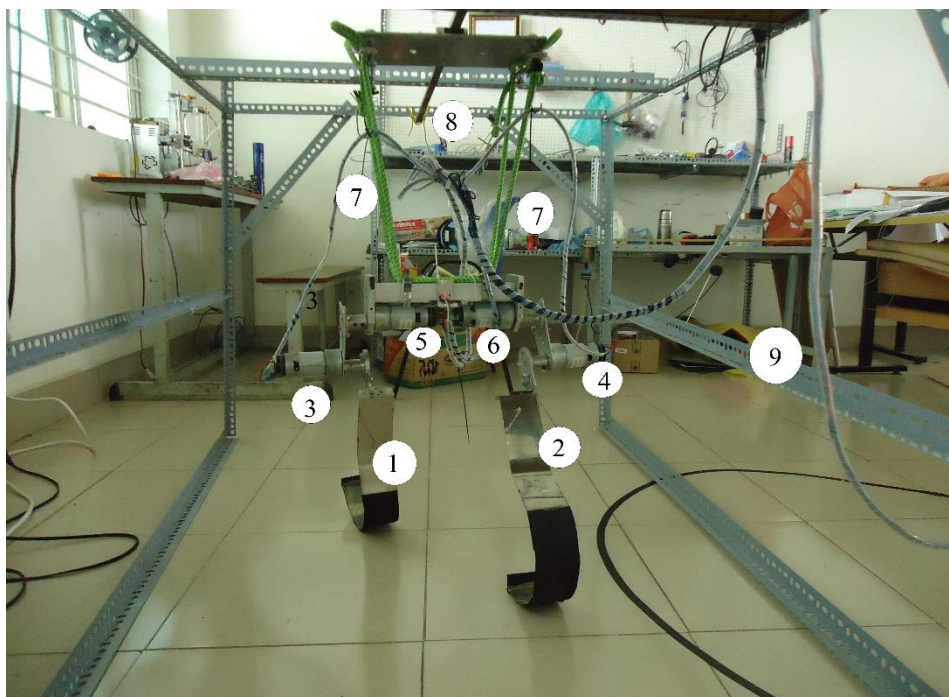


Figure 8.5: Imagine experimentală

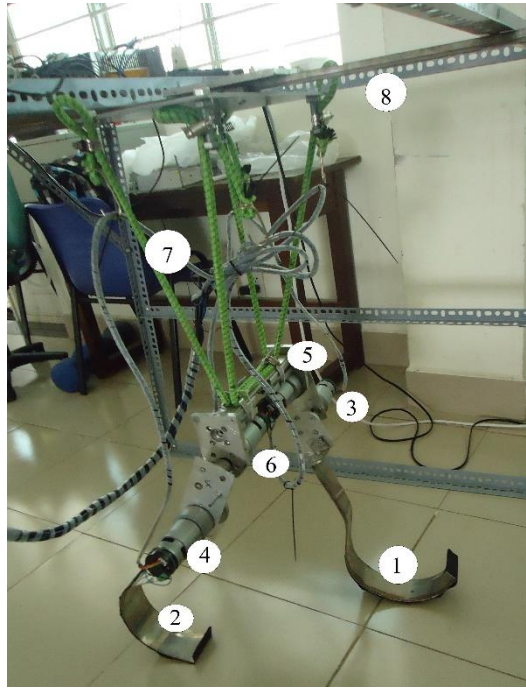


Figure 8.6: Imagine (foto) experiment

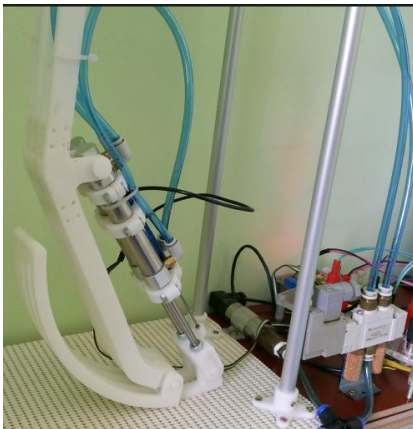
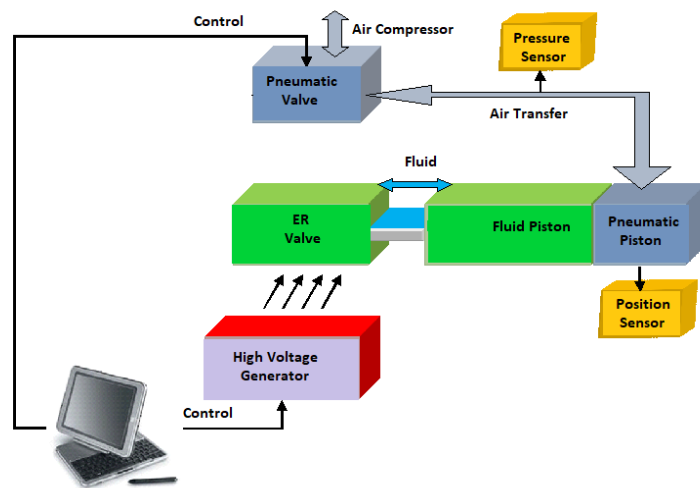


Figure 8.7: Platforma experimentală a arhitecturii de salt (Photo)

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- [6] US patent 5,701,965 Human transporter
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LIST OF PUBLICATIONS

International Journal

1. Huynh Xuan Dung, Huynh Duong Khanh Linh, Vu Dinh Dat, Nguyen Thanh Phuong, Nguyen Minh Tam, **Nguyen Van Dong Hai**, “Application of Fuzzy Algorithm in Optimizing Hierarchical Sliding Mode Control for Pendubot System”, *International Journal of Robotica & Management*, ISSN-L: 1453-2069; Print ISSN: 1453-2069; Online ISSN: 2359-9855 ,Vol. 22, Nr. 2, Dec-2017.
2. Nguyen Minh Hoang, Ngo Van Thuyen, Nguyen Minh Tam, Le Thi Thanh Hoang, **Nguyen Van Dong Hai**, “Desiging Linear Feedback Controller for Elastic IP with Tip Mass”, *International Journal of Robotica & Management*, ISSN-L: 1453-2069; Print ISSN: 1453-2069; Online ISSN: 2359-9855 , pp. 27-32, Vol. 21, Nr. 2, December-2016.
3. **Nguyen Van Dong Hai**, Huynh Xuan Dung, Nguyen Minh Tam, Cristian Vladu, Mircea Ivanescu, “Hierarchical Sliding Mode Algorithm for Athlete Robot Walking”, *Journal of Robotics*, ISSN: 1687-9600 (Print), ISSN: 1687-9619 (Online), Article ID

6348980, Hindawi, December-2017. DOI: doi.org/10.1155/2017/6348980 (ISI/ESCI/SCOPUS journal).

Link: <https://www.hindawi.com/journals/jr/2017/6348980/>

4. Nguyen Xuan Vu Trien, Le Thi Thanh Hoang, Nguyen Minh Tam, **Nguyen Van Dong Hai**, “Feedback Control Design for a Walking Athlete Robot”, Journal of Robotica & Management, ISSN-L: 1453-2069; Print ISSN: 1453-2069; Online ISSN: 2359-9855, Vol. 22, Nr. 1, June, 2017.
5. Mihaela Florescu, **Van Dong Hai Nguyen**, Mircea Ivanescu, “Output Track Controller with Gravitational for a Class of Hyper-Redundant Robot Arms”, Journal of Studies in Informatics and Control, Romania, 2015 (ISI/SCIE journal)
Link: <https://sic.ici.ro/output-track-controller-with-gravitational-compensation-for-a-class-of-hyper-redundant-robot-arms/>

International Conference

1. **Nguyen Van Dong Hai**, Mircea Ivanescu, Mircea Nitulescu, “Hierarchical Sliding Mode Control for Balancing Athlete Robot”, 21st International Conference on System Theory, Control and Computing (ICSTCC 2017), Sinaia, Romania, Nov-2017.
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